

- 1) A belt moves upward at velocity  $V$ , dragging a film of viscous liquid of thickness  $h$ , as shown in the figure. Near the belt, the film moves upward due to no-slip. At its outer edge, the film moves downward due to gravity. Assuming that the only nonzero velocity is  $v(x)$ , with zero shear stress at the outer film edge, derive a formula for (a)  $v(x)$ ; (b) the average velocity  $V_{avg}$  in the film; and (c) the wall velocity  $V_c$  for which there is no net flow either up or down.

(30)

**Solution:** (a) The assumption of parallel flow,  $u = w = 0$  and  $v = v(x)$ , satisfies continuity and makes the  $x$ - and  $z$ -momentum equations irrelevant. We are left with the  $y$ -momentum equation:

$$\rho \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = -\frac{\partial p}{\partial y} - \rho g + \mu \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

There is no convective acceleration, and the pressure gradient is negligible due to the free surface. We are left with a second-order linear differential equation for  $v(x)$ :

$$\frac{d^2 v}{dx^2} = \frac{\rho g}{\mu} \quad \text{Integrate: } \frac{dv}{dx} = \frac{\rho g}{\mu} x + C_1 \quad \text{Integrate again: } v = \frac{\rho g}{\mu} \frac{x^2}{2} + C_1 x + C_2$$

At the free surface,  $x = h$ ,  $\tau = \mu(dv/dx) = 0$ , hence  $C_1 = -\rho g h / \mu$ . At the wall,  $v = V = C_2$ . The solution is

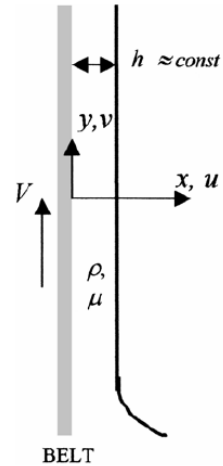
$$v = V - \frac{\rho g h}{\mu} x + \frac{\rho g}{2\mu} x^2 \quad \text{Ans. (a)}$$

(b) The average velocity is found by integrating the distribution  $v(x)$  across the film:

$$v_{avg} = \frac{1}{h} \int_0^h v(x) dx = \frac{1}{h} \left[ Vx - \frac{\rho g h x^2}{2\mu} + \frac{\rho g x^3}{6\mu} \right]_0^h = V - \frac{\rho g h^2}{3\mu} \quad \text{Ans. (b)}$$

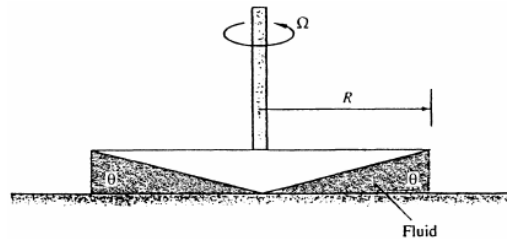
(c) Since  $h v_{avg} \equiv Q$  per unit depth into the paper, there is no net up-or-down flow when

$$V = \rho g h^2 / (3\mu) \quad \text{Ans. (c)}$$



- 2) The torque  $M$  required turning the cone-plate viscometer in figure below depends upon the radius  $R$ , rotation rate  $\Omega$ , fluid viscosity  $\mu$ , and cone angle  $\theta$ . Rewrite this relation in dimensionless form. How does the relation simplify if it is known that  $M$  is proportional to  $\theta$ ?

(30)



**Solution:** Establish the variables and their dimensions:

$$M = \text{fcn}( R , \Omega , \mu , \theta )$$

$$\{ML^2/T^2\} \quad \{L\} \quad \{1/T\} \quad \{M/LT\} \quad \{1\} \quad \mathbf{10}$$

Then  $n = 5$  and  $j = 3$ , hence we expect  $n - j = 5 - 3 = 2$  Pi groups, capable of only one reasonable arrangement, as follows: 5

$$\frac{M}{\mu\Omega R^3} = \text{fcn}(\theta); \quad \text{if } M \propto \theta, \quad \text{then } \frac{M}{\mu\Omega\theta R^3} = \text{constant} \quad \text{Ans.}$$

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- 3) The power  $P$  generated by a certain windmill design depends upon its diameter  $D$ , the air density  $\rho$ , the wind velocity  $V$ , the rotation rate  $\Omega$ , and the number of blades  $n$ . (a) Write this relationship in dimensionless form. A model windmill, of diameter 50 cm, develops 2.7 kW at sea level when  $V = 40$  m/s and when rotating at 4800 rev/min. (b) What power will be developed by a geometrically and dynamically similar prototype, of diameter 5 m, in winds of 12 m/s at 2000 m standard altitude? (c) What is the appropriate rotation rate of the prototype?

(30)

**Solution:** (a) For the function  $P = fcn(D, \rho, V, \Omega, n)$  the appropriate dimensions are  $\{P\} = \{ML^2T^{-3}\}$ ,  $\{D\} = \{L\}$ ,  $\{\rho\} = \{ML^{-3}\}$ ,  $\{V\} = \{L/T\}$ ,  $\{\Omega\} = \{T^{-1}\}$ , and  $\{n\} = \{1\}$ . Using  $(D, \rho, V)$  as repeating variables, we obtain the desired dimensionless function: **5**

$$\frac{P}{\rho D^2 V^3} = fcn\left(\frac{\Omega D}{V}, n\right) \quad \text{Ans. (a)}$$

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(b) At 2000 m altitude,  $\rho = 1.0067$  kg/m<sup>3</sup>. At sea level,  $\rho = 1.2255$  kg/m<sup>3</sup>. Since  $\Omega D/V$  and  $n$  are the same, it follows that the power coefficients equal for model and prototype:

$$\frac{P}{\rho D^2 V^3} = \frac{2700W}{(1.2255)(0.5)^2 (40)^3} = \frac{P_{proto}}{(1.0067)(5)^2 (12)^3}, \quad \mathbf{3}$$

*solve*    $P_{proto} = 5990 \text{ W} \approx \mathbf{6 \text{ kW}}$    *Ans. (b)*

**1**

(c) “Geometrically similar” means that  $n$  is the same for both windmills. For “dynamic similarity,” the advance ratio  $(\Omega D/V)$  must be the same:

$$\left(\frac{\Omega D}{V}\right)_{model} = \frac{(4800 \text{ r/min})(0.5 \text{ m})}{(40 \text{ m/s})} = 1.0 = \left(\frac{\Omega D}{V}\right)_{proto} = \frac{\Omega_{proto}(5 \text{ m})}{12 \text{ m/s}}, \quad \mathbf{3}$$

or:    $\Omega_{proto} = \mathbf{144} \frac{\text{rev}}{\text{min}}$    *Ans. (c)*   **1**